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COMPLETE SPECIFICATION.

Improvements in Motive Power Engines particularly adapted as Motors for Road Vehicles.

I, George Eli Whitney, of 88, Falcon Street, East Boston, County of Suffolk, Commonwealth of Massachusetis, United States of America, Manufacturer, do hereby declare the nature of this invention and in what manner the same is to be performed, to be particularly described and ascertained in and by the following statement:—

This invention has for its object the production of a novel engine of great power, economy, strength and simplicity, compact and of light weight, and particularly adapted for use as a motor for vehicles.

I have herein shown my invention as provided with means whereby the speed may be varied, the engine reversed, the supply of water regulated automatically, the sound of the exhaust deadened, and the exhaust steam dried so as to become invisible Defore it is discharged into the atmosphere.

It is essential in the production of a practical horseless vehicle propelled by steam, which for many reasons, I consider the most economical, and easily controllable motive power, that the motive mechanism should possess the greatest possible economy of fuel and water, which has to be carried in the vehicle, and the greatest possible lightness and compactness consistent with strength and durability.

In this connection I deem of great importance the arrangement of one or more cylinders and valve chests within a jacket connected with the boiler shell at one side, for thereby it becomes possible, by connecting the jacket with the interior of the boiler, to cause the hot water and steam when at boiler temperature to surround the cylinder and valve chest, thus acting to prevent condensation and to dry the steam in the cylinder, so that the exhaust steam contains less moisture and thereby decreases the waste of water, as well as aids in rendering the exhaust steam invisible, when further treated as hereinafter described.

The use of an upright boiler is considered by me essential in steam propelled read vehicles, as thereby considerable variation of water level, caused by passing over inequalities in the road, is of no consequence, and the arrangement of the engine on the side of the boiler affords at once lightness, strength, durability, and 30 compactness.

Fig. 1 is a front elevation, and partly in section, of an engine embodying my invention;

Fig. 2 is a partial top or plan view thereof, broken out and with the bonnet removed;

35 Fig. 3, in side elevation and partial vertical section, represents the engine shown in Fig. 1;

Fig. 4 is an enlarged detail view, partly in section, of the reversing mechanism;
Fig. 5, in elevation and partfully in section, represents the boiler, with the feed water controlling device;

40 Fig. 6, in side elevation, represents a throttle actuating device whereby a very slight movement of the throttle is effected.

[Price 8d.]

Fig. 7 shows the throttle in vertical section enlarged;
Fig. 8 is a section of the valve enlarged and detached;

Fig. 9 is an enlarged sectional view of the muffler for the exhaust steam, shown in elevation in Fig. 1;

Fig. 10 is a modified form of construction of the expansion member of the boiler, 5 to be referred to.

I have herein shown the boiler of my engine as of the upright tubular type, and comprising a preferably cylindrical steel shell A and water leg B surrounding the fire box or chamber B¹, the water leg forming the sides of the fire chamber.

The water leg is composed of concentric steel shells b, b¹, rivetted together at their 10 lower ends as at 5 and strengthened and stiffened in usual manner by stay bolts 6, a lower tube head b² being rivetted at 7 to the upper end of the inner shell b¹, and forming the top of the fire chamber.

A series of very thin, long copper fire tubes b^* are secured at their lower ends in the head b^2 , and at their upper ends in the upper or top head b^3 , see Fig. 3, the 15 latter being rivetted, as at 8, to the boiler shell A, a bonnet D surmounting and closing the top of the boiler, and connected with the downturned flue D¹⁰, for the escape of the products of combustion.

It is well known that the copper has a much greater rate of expansion than steel, and great trouble has been experienced, in using copper tubes of any length, in 20 firmly and rigidly securing them to the heads.

In fact, it is impossible to maintain copper tubes tight in boilers as now constructed, and consequently steel tubing is used, which is not only much more expensive than copper, but owing to the liability of steel to corrode if made very thin, the steel tubes must be made much thicker than is necessary with copper.

The increased thickness of steel tubes decreases the heating effect of each tube, and the boiler makes steam much more slowly.

The thinner copper tubes possess not only the non-corrosive and quicker heating qualities, but the further manifest advantage that a greater number of such tubes of a stated interior diameter can be inserted in a given boiler, thereby greatly 30 increasing the heating surface.

I have devised novel means whereby the copper tubes may be used without leakage, and owing to the thinness of the copper, a boiler made in accordance with my invention will steam very quickly and economically.

Between the lower end of the shell A and the upper end of the outer shell b of 35 the water leg, I have herein shown a thin resilient ring b^{10} preferably of steel, rivetted to the two parts at 9, 10, respectively, said resilient expansion ring being shaped in cross section substantially as shown, or two parallel flanges connected by a reverse-curved web, as clearly shown in Fig. 3.

The walls b, b^1 , of the water leg B are not connected at their upper ends, and as 40 the lower tube head b^3 forms a rigid part of the water leg, while the upper head b^3 forms a rigid part of the boiler shell A, the latter being supported by the expansion ring b^{10} , the latter permits the longitudinal expansion of the tubes b^* , and takes up the contraction, completely preventing disruption, or any leakage at the points where the tubes are secured to the heads.

It is obvious that the longitudinal expansion of the tubes may be provided for by making an integral portion of the boiler shell serve as the expansion member, as for instance by making an annular bend in the shell between the portions to which the tube heads are secured, one form of such modification being shown in Fig. 10.

A main crank shaft C is supported in suitable bearings C* secured by 50 brackets 120 to the water leg, the preferably quartering cranks C¹ and C² being connected with the piston rods P¹, P², by connecting rods c¹, c², cross heads c³, c⁴, rigidly secured to the piston rods sliding on vertical guides c⁵, c⁵, secured by brackets 12 and 13 to the shell A and water leg respectively.

Inasmuch as the expansion member of the boiler is located between the crank 55 shaft bearings C*, to which the lower ends of the brace rods a are attached, and the rigid points of attachment of their upper ends, the variation in the distance between

these points of attachment, due to action of the expansion member, is provided for by the brackets 120, which while rigid laterally are sufficiently flexible in a vertical direction.

The brackets 12 and 13 which support the guide rods c5 and c6 in like manner

5 yield sufficiently to prevent any distortion of the said guide rods,

The two cylinders P, P* are mounted in and supported by a jacket E preferably comprising flanged top and bottom members e, e⁴, rivetted to the boiler shell, as herein shown, and a vertical side wall e², rivetted at top and bottom to said members and at its ends to the shell A, making a rigid and strong, yet light, support.

Referring to Fig. 1, it will be seen that the flanged ends of the cylinders are screwed to the members c and c1, which have openings therein to receive the flanged

cylinder heads p, p1, bolted in place.

The vertical brace rods a, a¹, passing through the jacket bottom e¹, and secured to the cylinders and to the crank shaft bearings C* hold the crank shaft in

15 position.

Referring to Fig. 3, openings 3 are shown in the shell A, which communicate with the interior of the jacket E, so that the hot water and steam from the boiler will surround the cylinders and their valve chests, of ordinary construction, thus preventing cooling by radiation and condensation and maintaining the cylinders at 20 high temperature, and greatly increasing the economy of the apparatus, by drying the steam in the cylinders.

The brace rods a are arranged in pairs, and each pair has rigidly secured thereto outwardly extended stands F which provide bearings for the valve actuating

shaft f*, shown separately in Fig. 4.

This shaft has cranks f¹, f², for the connecting rods f³, f⁴, jointed in turn to the valve stems v¹, v², working in the valve chests V.

The shaft f^* is elongated at one end, as at f^5 , Fig. 4, to receive a sleeve f^6 , longitudinally movable thereon and connected to and to rotate with the shaft by a key f^7 , said sleeve having, as herein shown, a cam groove f^8 in its outer surface.

A sprocket wheel f^9 has an elongated hub f^{10} which receives the sleeve f^6 , a lug or projection f^{12} on the hub entering the cam groove f^8 , said sprocket being connected by a suitable sprocket chain f^{13} with a sprocket wheel e^8 on the crank shaft C.

Rotation is thus transmitted from the latter to the valve actuating shaft f*, which operates the valves, and by moving the sleeve f⁶ by any suitable means in 35 or out on the valve shaft, the latter through the projection f¹² and cam groove f⁸ acts to reverse the direction of rotation of the shaft and thereby change the valves to reverse the engine.

In motor vehicles particularly, passing as they must over various kinds and varieties of road-beds, it is evident that the boiler will be subjected to more or less 40 vibration or jarring, which has a tendency to cause feaming, and should priming take place, the water passing over into the cylinder will usually terminate in an

accident to the mechanism.

Instead, therefore, of taking the boiler steam from a single point, I connect the steam pipe d, Figs. 2 and 3, with a plurality of tubular branches d¹⁹ which enter the steam space at different points, and I thus almost entirely prevent the outflow of the steam from taking water over into the engine cylinder, even should foaming take place under ordinary circumstances.

To still further guard against the entrance of water to the steam pipe, I place a suitable baffle plate d²⁰ below the inner end of each of the various branches d¹⁰, and by the distribution of the steam outlets over different portions of the steam space of the boiler, assisted by the baffle plates, I am enabled to operate the engine and boiler with entire safety and without fear of priming even should considerable feaming occur.

The steam pipe d is provided with a throttle valve having a stem t, said pipe 55 leading from the top of the steam space of the boiler to the valve chests, as described, and making an independent steam connection between the boiler and

The jacket E is provided with a man-hole E1 and cover plate E2, by which access

may be had to the interior of the jacket when necessary.

By referring to Figs. 1 and 2, it will be seen that when the cover plate E² is removed, direct access may then be had to the valve chests to examine the same or to adjust the valves even while steam is up and full pressure on in the boiler, 5 for the steam in the jacket is not admitted to the space between the valve chests thus left open by the removal of the cover.

So, too, if for any reason it is necessary to have access to the cylinder while the steam is up, the cylinder heads may be removed as there is no direct communication between the jacket and the cylinders, the steam admitted to the valve chests and 10

cylinders being controlled by the throttle.

The power is transmitted from the crank shaft C in any suitable manner, and as herein shown by a sprocket wheel C³, if it is desired to apply the engine to a vehicle to drive the same.

An upright 70, see Fig. 6, is erected on the upper end of the cylinder jacket on 15 which is fulcrumed at 71 a lever 72, jointed at one end to the throttle valve stem t, the other end of said lever being jointed at 73 to one arm 74 of a toggle, the other arm 75 being fulcrumed at 76 to a fixed upright 77.

By straightening or breaking the toggle, the lever 72 will be rocked to move the throttle, and when the toggle is nearly straight, very slight movement will be 20 imparted to the throttle valve so that it may be regulated with great nicety, while

its movement is greater when the toggle is bent more.

A link 78 connects the joint of the toggle with an arm 79 on a rock shaft 80 having a bearing in the upright 70, and said rock-shaft has attached to it an arm t¹³, operated by a link t¹³, whereby the rock-shaft is turned by a suitable 25 connection with a controlling device, not herein shown.

Figs. 7 and 8 show details of the throttle. It consists essentially of an outer

case marked d in which is mounted a valve seat t17.

The valve stem t within this case has on its end a valve composed of a flat faced disc t¹⁸, having on its side or edge guiding projections to bear against the interior 30 of the case.

The valve is closed tight by the contact of the flat seat and face, and opened by

separating them.

In order to be able to separate the flat seat and face without a powerful rush of steam entering through the throttle into the cylinder and thus making the vehicle start with a jump, when the engine is applied to a vehicle to provide the motive power therefor, I place on the centre of the face of the valve a conical projection t¹⁹, which when the valve is closed, stands in and substantially fills the upper part of the opening in the valve seat.

By the addition to the valve of the said projection, it is possible when opening the 40 valve, by separating the flat face and seat, to leave a much smaller passage for the steam than would otherwise be left, thus to ensure the very gradual and slow passing of the steam through the seat to the cylinder, or, in other words, the said cone so fills the seat that the steam can issue but slowly until after the flat part of

the valve has been moved for a considerable distance from the said seat.

By the mechanism shown in said Fig. 6, the throttle is under complete control, but to attain greater refinement of movement of the said throttle, which is necessary in order to start a motor vehicle slowly, or to move very slowly as may be necessary with such a vehicle in crowded streets, I also make use of the mechanism shown in Figs. 7 and 8.

In Fig. 1 the exhaust pipe E¹⁰ leads from the valve chests to the flue D¹⁰ on the

exterior of the boiler shell.

Referring to Fig. 5, I have shown a pipe 40 opening into the top of the boiler, said pipe at its other end opening into one end of a chest or chamber 41, arranged horizontally and at the normal water level.

The other end of the chest is connected by a return pipe 42 with the lower portion of the water leg B, so that the water may freely find the same level in the

pipe 40 that is maintained in the boiler, and while freely finding such level the water in these pipes and in the chamber 41 does not circulate otherwise than very slowly,-it remains practically dead, and therefore at a much lower temperature than the water and steam in the boiler.

The chest is provided with a gland or stuffing box 43 through which passes an expansion rod 44, secured at its inner end, and at its free end attached to a valve 45 in the feed-water supply pipe 46, which connects the water tank with the boiler at a suitable point.

A supply pump constantly in operation is attached to the feed water pipe 46 at

10 a suitable point between the valve 45 and the boiler.

Now when the water is at the proper level, and therefore standing in the chest 41, the expansion rod will maintain the valve 45 closed, but when the water level falls, steam passes from the boiler into the chest, and by its greater temperature causes rod 44 to expand and open the valve, admitting an additional supply of feed-water 15 to the supply pump and boiler, the valve being automatically shut when the water

is raised high enough to fill the chamber 41.

The expansion rod may be made of aluminium which I find will give highly satisfactory results, or it may be made of other metal having the requisite degree of

By means of valves at 40° and 42°, the automatic controller for the feed-water

may be shut off from the boiler.

It is obvious that the feed water regulator herein described, may be applied to any type of steam boiler, wherein it is desired to automatically maintain a

substantially constant water level.

I prefer to heat the boiler by naptha, kerosene, or other suitable liquid fuel, supplied to the combustion chamber or furnace within the water leg B, and provided with suitable burners, not shown, and I have herein shown a feed regulator for the fuel, see Fig. 5, the same not being herein described in detail as it forms no part of this invention.

I prefer to mount the engine and boiler on a road vehicle in such a manner that

the same will be partly within and partly below the body thereof.

For the purposes of obtaining greater comfort to the occupants, compactness, improving the looks of the vehicle to which my present invention may be applied, and making it less unlike ordinary vehicles, I construct the flue D10 as in Fig. 1, 35 with its discharge opening at the lower end of the boiler, so that the slight gaseous products of combustion may pass off beneath the vehicle out of the sight and smell of the occupants.

In order to induce a downward draft, I insert a downwardly pointed steam jet D14

connected with the boiler, in the flue D10 at about the point shown in Fig. 1.

Now, in using a steam motor for such vehicles, it must be born in mind that a cloud of escaping steam from the exhaust, especially if accompanied with the puffing noise usual in the exhaust of high pressure engines, would be very objectionable, mainly on the ground that it would be unsightly and would frighten passing horses.

By the prevention of priming in the manner described and by exhausting the steam in a substantially dry state from the cylinder, due to the jacket described, there is very little moisture to be condensed, and the exhaust is further dried by its entry into the flue D10, preferably at a point as near the top as possible, where it is acted upon by the hot gaseous products of combustion, so that there is practically 50 under ordinary circumstances no visible exhaust issuing from the flue.

The puffing noise, however, must be obviated by other means, and for this purpose I have devised a muffler and pressure equalizer, interposed in the exhaust pipe E10 between the valve chest and the flue D10, the muffler serving to completely deaden any sound from the exhaust.

The muffler is shown in section in Fig. 9, as a metal cylinder M into the ends of which the exhaust pipe E10 is tapped, and the cylinder is filled with fibrous material M*, preferably curled hair.

An envelope M1 of wire gauze preferably surrounds the fibrous material, maintaining it in place in the cylinder and preventing its passage into the exhaust

In actual practice the exhaust steam passes in successive puffs from the cylinder to the muffler, and the fibrous material M* seems to so break up or disintegrate the 5 same that it issues therefrom into the flue D10 in a steady, continuous stream

without the least noise.

The passage of the steam from the exhaust into the flue is thus continuous, and thereby the hot gases of combustion coming down the flue have time to dry and evaporate the exhaust steam or particles of water condensed therefrom, rendering it 10 invisible.

This the hot gases would not have so good an opportunity of doing if the steam or particles of water condensed therefrom were forced into the fine in successive

intermittent puffs.

My invention is not restricted to the precise construction and arrangement of the 15 various parts hereinbefore set forth, as obviously, changes in the construction and re-arrangement of parts may be made without departing from the spirit and scope of my invention.

Having now particularly described and ascertained the nature of my said invention, and in what manner the same is to be performed, I declare that what 20 I claim is:-

In a motive power engine particularly adapted as a motor for a road vehicle.

1. A steam boiler comprising a fire chamber, water leg, and shell, the water leg having inner and outer walls joined at their lower ends and forming the sides of the fire chamber, the outer wall being a continuation of said shell, a lower tube head 25 attached to the inner wall of the water leg and forming the top of the fire chamber, an upper tube head attached to the upper portion of said shell, fire tubes constituting a rigid connection between said heads within the boiler shell, and communicating with the fire chamber, the tubes being composed of a material having a different rate of expansion from that of the material of the shell, and an 30 expansion member interposed between the parts of the boiler which support the tube heads, whereby the difference of expansion between the parts joining the peripheries of the said heads and the tubes joining the interior portions of the heads is permitted without injury.

2. In an apparatus of the class described, an upright boiler, an engine mounted on 35 the shell thereof, a jacket surrounding the engine cylinder and secured to the boiler shell, said jacket communicating with the interior of the boiler, and an independent steam supply connection between the cylinder and the boiler.

3. A steam boiler shell, an external chamber or jacket secured thereto, the boiler shell forming the rear wall of the jacket and having openings thereinto, respectively 40 above and below the normal water level, and a cylinder within the jacket and . detachably secured to the ends thereof, and an independent steam supply connection between the cylinder and the boiler.

4. A boiler shell, an exterior jacket, a cylinder and valve chest supported by and enclosed in said jacket, a steam supply connection between said chest and boiler, 45 and an independent connection between the interiors of the shell and jacket, whereby the hot water and steam may circulate around the cylinder and valve

5. In an apparatus of the class described, a boiler, an engine mounted thereon wholly exterior thereto, a jacket secured to the boiler shell and communicating 50 with the interior thereof, the engine cylinder being attached to the jacket ends, and cylinder heads externally removable and adapted to be secured to the cylinder ends.

 The boiler shell having attached jacket ends e and e¹, and the cylinder having its flanged heads and ends bolted to said jacket ends, a side wall interposed between the jacket ends and connected therewith and with 55 the boiler shell, steam tight, thus forming a jacket enclosing said

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cylinder, the interior of the jacket communicating with the boiler above and below the normal water level, whereby hot water and steam therefrom circulate about and surround the cylinder, and an independent steam supply connection between the cylinder and the beiler.

5 7. A tubular boiler comprising a shell and tube heads secured to the upper and lower portions thereof, tubes rigidly attached to the heads, and having a different rate of expansion from the shell, an expansion member for the shell, between the tube heads, an engine mounted exteriorly on the shell, and connections between said engine and shell laterally rigid and adapted to yield vertically to permit the

10 operation of the expansion member of the boiler.

8. Propelling mechanism for road vehicles, comprising an upright boiler, an engine mounted externally on said boiler, its cylinder and main shaft, a jacket surrounding and supporting the cylinder and secured to the boiler, bearings for the main shaft, a support for said bearings laterally rigid and adapted to yield 15 vertically, and secured to the boiler shell, and upright brace rods rigidly connecting said bearings and cylinder.

In an upright, tubular steam boiler, the throttle, and a plurality of tubular branches in the upper part of the boiler leading to a single inlet to the throttle from different points of the upper tube head and communicating with the steam

20 space, whereby passage of water to the throttle is prevented.

10. In an upright, tubular steam boiler, the throttle, a plurality of tubular branches leading to a single inlet thereto from different points of the steam space, the inner ends of said branches being held in the upper tube head, and baffle plates located below the tube head and the inner ends of the branches, to thereby prevent 25 passage of water with the steam to the throttle.

11. In a steam engine, its throttle, a lever connected therewith and having a fixed fulcrum, a toggle one arm of which is jointed to said lever, the other arm having a fixed fulcrum, and means connected with the joint of the toggle, to straighten or break the latter and thereby rock the the throttle actuating lever.

30 12. A steam boiler and engine having a steam drying apparatus consisting of a cylinder and a valve chest enclosed in a steam jacket mounted upon and connecting with the interior of the boiler, whereby the steam in the cylinder and valve chest is kept hot and dry, a flue for the escape of the products of combustion, a steam exhaust pipe leading from the cylinder and opening into said flue, and a muffler 35 and pressure equalizer in said pipe between the cylinder and the flue, whereby the exhaust of the steam into the flue is made continuous instead of intermittent, and the steam, or particles of water condensed therefrom is dried and made

13. An upright boiler, an external jacket or chamber secured thereto, a cylinder 40 within the jacket, a throttle to control the steam supply for the cylinder, means to draw steam from a plurality of points in the boiler steam space for the throttle supply, and to prevent passage of water thereto, and a connection between the jacket

and boiler.

14: -An upright boiler, an external jacket or chamber secured thereto, a cylinder 45 within the jacket, a throttle to control the steam supply for the cylinder, means to draw steam from a plurality of points in the boiler steam space for the throttle supply and to prevent passage of water thereto, and connections between the jacket and boiler, respectively above and below the water level, whereby hot water and steam may surround the cylinder and valve chest.

50 15. An upright boiler, an engine mounted exteriorly thereon, an external chamber or jacket enclosing and supporting the cylinder and valve chest of the engine and located near the upper end of the boiler shell, a crank shaft supported on the lower portion of said boiler, a throttle for the cylinder, means to draw steam from a plurality of points in the boiler steam space for the throttle supply and to

55 prevent passage of water thereto, and a connection between the interior of the boiler and the jacket. 16. A boiler shell, an exterior jacket, a cylinder and valve chest supported upon

and enclosed in said jacket, a connection between the interior of the shell and the jacket a throttle to control the supply of steam to the valve chest and cylinder, and means whereby access may be had to the varye chest through the side wall of

17. A boiler, an engine mounted thereon wholly exterior thereto, a jacket secured 5 to the boiler shell and communicating with the interior thereof, the engine cylinder being attached to the jacket ends, cylinder heads externally removable and adapted to be secured to the jacket ends and eylinder a valve chest within the jacket, a removable cover for an opening in the jacket adjacent the valve chest, and a throttle to control the entrance of steam to the valve chest and cylinder whereby 10 access may be had to either the valve chest or cylinder while the steam in the boiler and in the jacket surrounding the cylinder and valve chest is under pressure.

18. A boiler shell having an external jacket or chamber, attached to the side thereof and comprising jacket ends flanged at their front, side and rear edges, the rear flanges being attached to the boiler shell, and an interposed wall between said 15 jacket ends and connected with the front and side flanges thereof, the ends of said walls being flanges and attached thereat to the boiler shell, the whole forming a steam-tight jacket a cylinder supported and enclosed within said jacket, and a communication between the interiors of the boiler and jacket.

Dated the 21st day of March 1898.

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